

Conceptual Design and Development of an Indirect-cooled Superconducting Helical Coil in the FFHR

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The force-free helical reactor (FFHR) is a conceptual design of a heliotron fusion reactor being developed at the National Institute for Fusion Science. All the coils in the FFHR are made of superconductors. Several cooling schemes have been proposed for the helical coils; of these, indirect cooling is considered a good candidate. In this study, we investigated the possibility of using an indirect-cooled superconducting magnet for the FFHR. In parallel with this design study, we developed Nb₃Sn superconductors, jacketed with an aluminum alloy, for use in an indirect-cooled magnet. The results of performance tests of a sub-scale superconductor showed good feasibility for application in the FFHR helical coil. Stress distribution in the helical coil was also analyzed, and the stress and strain were confirmed to be within the permissible range.

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1. Introduction

Experimental results of the Large Helical Device (LHD) have revealed that an LHD-type helical reactor is well suited as a demonstration device for a fusion power plant [1]. The force-free helical reactor (FFHR) is a conceptual design of an LHD-type heliotron fusion reactor being developed at the National Institute for Fusion Science [2, 3]. The FFHR's magnet system includes one pair of helical coils and two pairs of poloidal coils. All the coils are made of superconductors. Several cooling schemes, such as forced flow and indirect cooling, have been proposed for these superconducting helical coils. The former, with a cable-in-conduit conductor (CICC), has been chosen in many large-scale experimental fusion magnets, such as the poloidal coils of the LHD, the main coils of ITER, Wendelstein 7-X, and JT-60SA, because of its mechanical strength and electrical/thermal stability. However, forced flow cooling with a CICC requires a circulation pump to maintain pressure to a supercritical region of the coolant. Therefore, the length of the cooling path can be limited. For instance, the maximum cooling length in the ITER specification is 390 m. Modifications of the cooling or winding method are needed for a magnet whose conductor length is several times that of an ITER-class magnet. On the other hand, indirect cooling solves this problem because the electrical and cooling paths can be separated. Furthermore, an indirect-cooled superconducting magnet is considered to have higher mechanical rigidity, since its structural components, such as the superconducting strands, jacket, insulators, cooling panels, and coil

case, are completely in contact with each other.

In this study, we investigated the possibility of using an indirect-cooled superconducting magnet for the FFHR. In parallel with this study, we developed Nb₃Sn superconductors, jacketed with an aluminum alloy, for use in the indirect-cooled magnet. The “react-and-wind” process can be applied in a large superconducting coil using this type of superconductor, since the jacketing can be done after heat treatment of the superconducting strands by friction stir welding (FSW [4]). The details of sample conductor development and test results of a sub-scale superconductor are also given.

2. Structure of the Coil

Figure 1 shows a schema of the cryogenic components in the FFHR. The major and minor radii of the helical coils

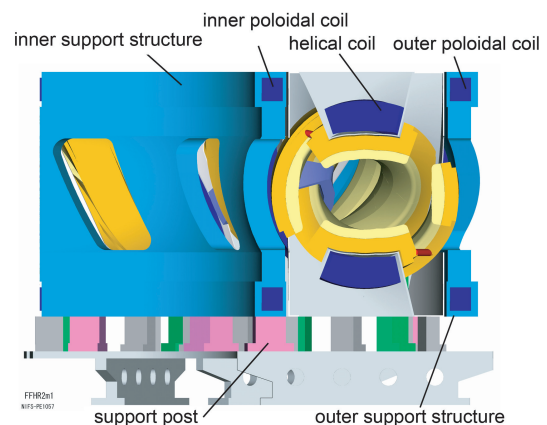


Fig. 1 Schema of the cryogenic components in the FFHR.

are approximately 14–16 m and 4 m, respectively. The total stored energy of the coils is 120 GJ. The electromagnetic force generated by these coils is sustained by inner and outer support structures. The magnetic field at the plasma center is 6.18 T. The cross-sectional dimensions of the helical coil were determined by considering the geometry of the plasma-facing components. Figure 2 shows a conceptual design of the cross section of the helical coil, which is a rectangle 1.8 m wide and 0.9 m high. There are 432 superconductors (36 turns, 12 layers) made of Nb₃Sn and a jacketing material. An aluminum alloy was chosen as the jacketing material because it offers high thermal conductivity and mechanical strength. The cooling panels were placed at every two or four turns of the winding. Each cooling panel consists of two parts: a cooling module and a case section. The cooling module is connected to coolant plumbing outside the coil cross section. The case section has to be built continuously with the superconductor so it will be a strength member, whereas the cooling module can be divided into several parts along the winding direction. The total thickness of each cooling panel is 75 mm, and the superconductor is indirectly cooled by these panels. The coil is wound along the stainless steel (SS) coil case and covered with a lid. An LHD-type helical reactor does not require plasma current, so little AC loss occurs in the magnet. The heat load to the coil during reactor operation comes mainly from nuclear heating. Takahata *et al.* calculated the elimination of this steady-state heat load and showed that an aluminum jacket superconductor with a cooling panel could resolve this issue [5].

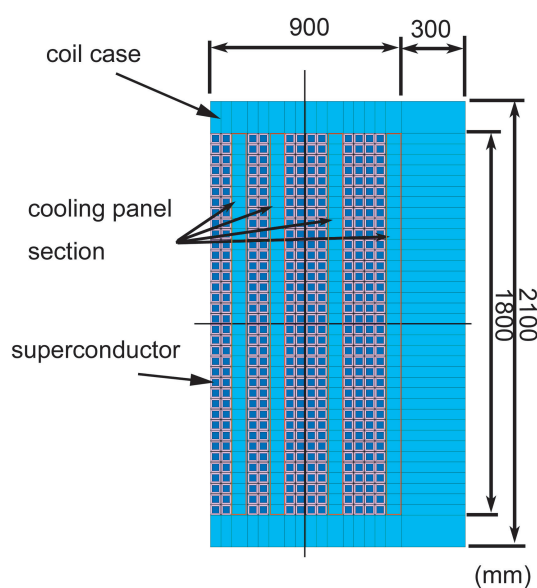


Fig. 2 Cross-sectional view of the conceptual design of the indirect-cooled helical coil.

3. Development of the Aluminum Alloy Jacketed Superconductor

3.1 Specification and fabrication process

The fundamental geometry of the superconductor is a 50 mm square, including insulation. Since the maximum magnetic field at the coil region is around 13 T, Nb₃Sn wires can be used. The operating current is 100 kA, and the overall current density is 40 A/mm². Since the melting point of aluminum alloy (933 K) is lower than the heat treatment temperature of the Nb₃Sn wires (1000 K), jacketing must be done after heat treatment of the wires. We developed a conductor fabrication process, using an FSW technique that uses friction heating, to avoid a temperature increase in the welding region. Since FSW is a solid-state joining process, the temperature at the welding region is below the melting temperature of the material. In the case of the aluminum alloy, the temperature does not exceed 900 K. The superconducting wire is embedded in the aluminum alloy jacket with a solder material, and the lid is welded by FSW. The solder can be melted by heating from the FSW, and it fills the void around the superconducting wire. A prototype 10-kA-class 17-mm-square superconductor was made to demonstrate the fabrication process and the performance of the conductor. It showed a 19 kA transport current at 8 T and confirmed that although some degradation occurred in the critical current, it was not due to the fabrication process, but the difference in thermal contraction between Nb₃Sn and aluminum alloy under a change from room temperature to 4 K [5].

3.2 Reduced-size sample test

To confirm the thermal influence of the FSW and examine an instance of bending deformation, 4.7-kA-class superconductors, made of Nb₃Sn cable and aluminum alloy jackets using the same production process as the 10-kA-class sample, were manufactured. The packing factor of the superconductor inside the aluminum jacket was increased from 60% to 80%. Figure 3 shows a cross-sectional view of the sample conductor and its dimensions. The following two samples were tested: (1) without bending and (2) bent once along a ring with a radius of 150 mm and then bent back to the original straight shape (R150S).

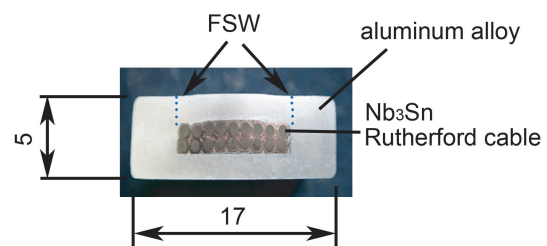


Fig. 3 Photo of the cross-section of reduced-size 4.7-kA-class sample superconductor.

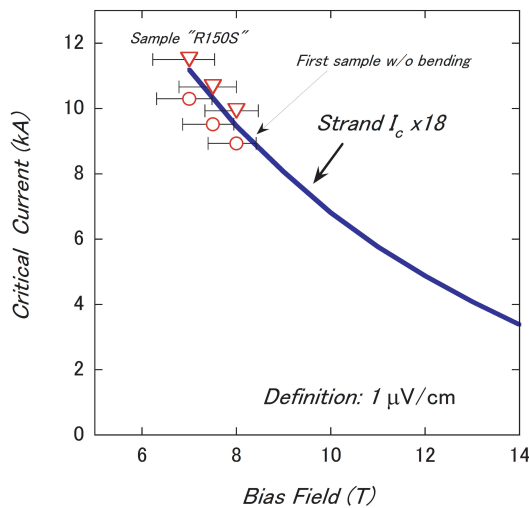


Fig. 4 Critical currents of the reduced size superconductor, with and without bending.

Figure 4 shows the experimental results of current-carrying capacity tests. Open symbols indicate the critical current (definition; 1 μV/cm) of the test conductor at 4.4 K; error bars represent the maximum and minimum magnetic field inside the conductor. The solid line indicates the critical current of each strand multiplied by 18 (the number of strands). We succeeded in carrying a current of 11 kA at 8 T with the sample R150S. This confirms that the critical current was not affected by bending deformation. Furthermore, the critical current of sample R150S might be increased by a pre-bending effect [6].

4. Comparison with CICC

Here we simply compared the apparent rigidity of the indirect-cooled and CICC-type superconductors. Figure 5 shows models of the superconductors. The indirect-cooled superconductor has a 50-mm-square shape and a 32 mm square Nb₃Sn superconducting region filled with solder. The ratio of the superconductor to the solder is 8 : 2. The superconductor includes an 18-mm-thick aluminum alloy (6061 T6) and 1-mm-thick insulation. The CICC type has 90 kA of operating current with 480 superconductors made of Nb₃Sn. The conduit is 1.6 mm thick, and the conductor is embedded in the internal plate. Both components are made of SS. There is an insulator between the conduit and the internal plate.

The longitudinal rigidity was estimated according to the rule of mixture, using the area fraction of each structural component. The transverse rigidity was calculated by modeling each conductor type with a finite element method (FEM) model. In this case, the plane strain model was adopted, and the rigidity was calculated from the result of the reaction force against the forced displacement at the top of the conductor. In the indirect-cooled type, the material properties of the superconducting region were se-

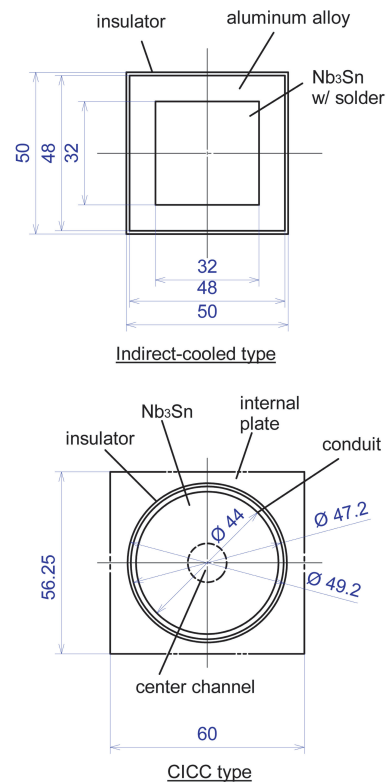


Fig. 5 Rigidity evaluation model for the indirect -cooled (upper) and CICC (lower) superconductors.

Table 1 Material properties of components.

	Young's modulus (GPa)	Poisson's ratio	note
Nb ₃ Sn	100	0.3	
Aluminum	77	0.327	6061-T6
Insulator	80	0.3	Alumina w/ epoxy
Stainless steel	208	0.284	SS316
Solder	40	0.3	Pb free

lected according to the rule of mixture. On the other hand, the superconducting region in the CICC type was assumed to make no contribution to the mechanical rigidity in the transverse direction. The other components were treated as isotropic materials. The material properties of the components at a cryogenic temperature (4 K) [7–9] were used in the analytical model. The material properties used in the calculation are shown in Table 1.

The longitudinal rigidity of the indirect-cooled and CICC superconductors were estimated at 82 and 109 GPa, respectively. The former coil has a cooling panel that also contributes to coil rigidity. If the cooling panel has a longitudinal rigidity of 163 GPa, the indirect-cooled coil can provide reasonable overall rigidity compared with the

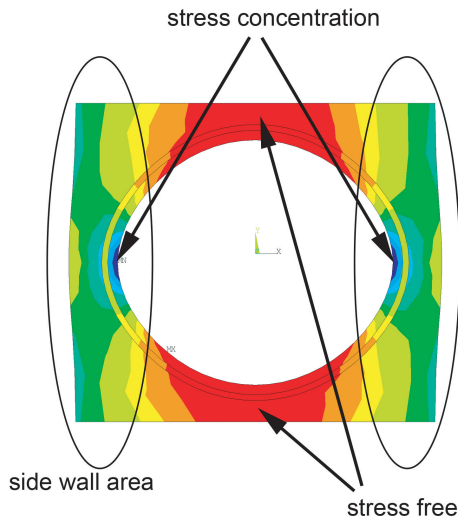


Fig. 6 Compressive stress distribution in the CICC.

CICC coil. Assuming that the case section in the cooling panel is made of SS316 and no stress concentration occurs, 20 % of the total cooling panel area can be used for the cooling module. The transverse rigidity of the indirect-cooled and CICC types were 79 and 56 GPa, respectively. Note that stress concentration may occur in the CICC, as shown in Fig. 6. The CICC with an internal plate resists transverse deformation only with the side wall area of the conduit and the internal plate.

5. Coil Rigidity Evaluation

5.1 Analytical model

The helical coil of the FFHR has a three-dimensional structure, with a change in its curvature in the toroidal angle. It is believed that a circular coil with an average curvature similar to that of an actual helical coil can be used to estimate the mechanical behavior of the coil [10]. We calculated the stress and strain distributions inside the coil to confirm the stress and strain levels. The average radius of curvature of the helical coil was 5.5 m at the center of its cross section. The cross-sectional structure of the helical coil, shown in Fig. 2, was used to create the FEM model. The radius from the central axis to the center of the coil cross section was set at the average curvature of the helical coil. The insulator used in the superconductors was assumed to be made of alumina ceramics and resin. An SS coil case, with a thickness of 300 mm at the top and 150 mm on both sides of the coil section, was used.

Because of an interaction between the magnetic field and the current flow, the magnetic field parallel to the central axis produced a hoop force, while the radial field produced an overturning force. The electromagnetic force was applied as the body force by multiplying the current density and magnetic field in every superconducting region to ensure that the electromagnetic force was precisely applied to the coil. The electromagnetic force considered here was

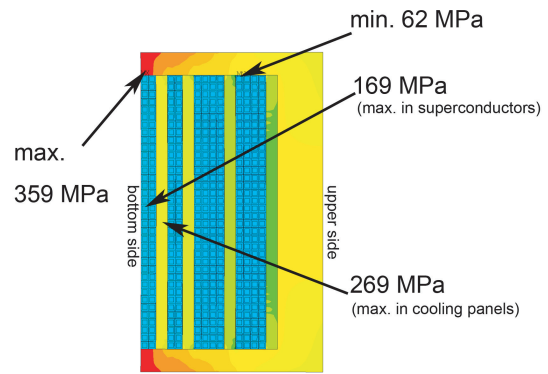


Fig. 7 Hoop stress distribution by the radial electromagnetic force.

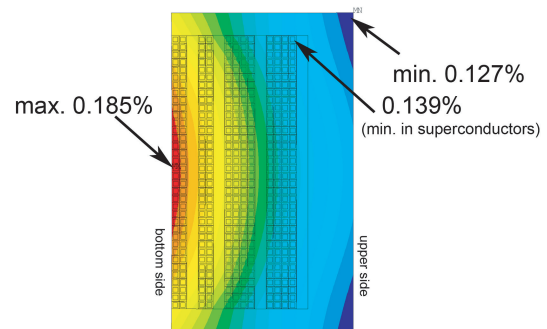


Fig. 8 Hoop strain distribution by the radial electromagnetic force.

in the radial direction of the circular coil, since it generated a hoop force inside the coil. The hoop force is more effective for the superconductor than the overturning force at a point of strength of the coil structure. Although the magnetic field intensity was different at each cross section, an averaged magnetic field was applied at each superconducting position along the circumference. Furthermore, a constant value was added to the averaged magnetic field so that the total hoop force in the cross section was equal to the maximum overall hoop force. ANSYS version 11.0 was used, and a three-dimensional axisymmetric solid element was adopted.

5.2 Results

The material properties were set using the values described in section 4. The cooling panel was assumed to have 80 % of Young's modulus for SS316. Figures 7-10 show the results of hoop force analysis with respect to the hoop stress distribution, hoop strain distribution, radial displacement distribution, and transversal strain distribution, respectively. A maximum hoop stress of 359 MPa appeared in the side wall of the coil case. In the coil winding and cooling panel sections, the maximum stress was 169 and 269 MPa, respectively. The strain from the hoop force was 0.185 % at the bottom center of the superconductor. The components in the coil were subjected to compressive

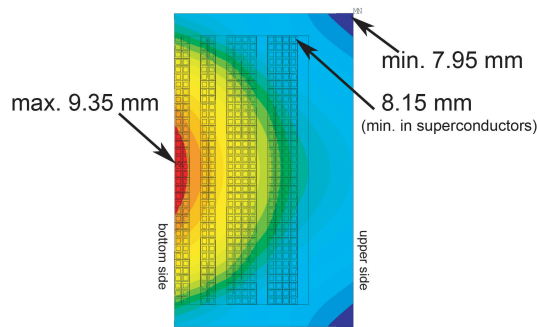


Fig. 9 Radial displacement inside the coil by the radial electromagnetic force.

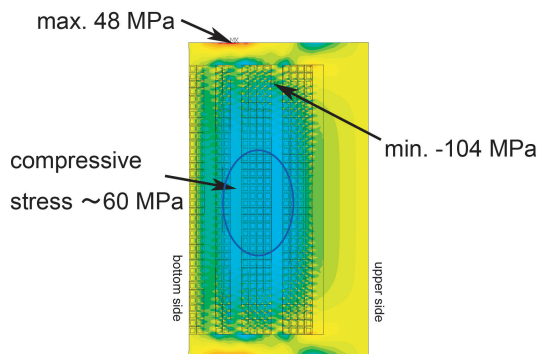


Fig. 10 Transversal stress distribution by the radial electromagnetic force.

sive stress toward the coil center region. The maximum in-plane shear stress in the insulation was 32 MPa. All stress and strain levels for each component were within the permissible range.

6. Conclusions

In conceptual design studies of the FFHR, indirect-cooled superconducting helical coils have been proposed. Aluminum-alloy-jacketed Nb₃Sn superconductors with a

cooling panel can prove the feasibility of this approach. The following results were obtained in this study: (1) A reduced-size sample of the aluminum-alloy-jacketed Nb₃Sn superconductor showed good performance, and the critical current did not degrade with bending. (2) The cooling panel requires a longitudinal rigidity of 163 GPa to provide the same rigidity as a helical coil using CICC. (3) The indirect-cooled superconductor has a much higher transverse rigidity than the CICC. (4) Stress and strain distributions in the indirect-cooled helical coil, investigated by the FEM model, were confirmed to be within the permissible range.

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